When Helical gears are operated on other than Parallel shafts, the tooth load is concentrated at a point, with the result that very small loads produce very high pressures. The sliding velocity is usually quite high and, combined with the concentrated pressure, may cause galling or excessive wear, especially if the teeth are not well lubricated. For these reasons, the tooth load which may be applied to such drives is very limited and of uncertain value, and is perhaps best determined by trial under actual operating conditions. If one of the gears is made of bronze, the contact area and thereby the load carrying capacity, may be increased, by allowing the gears to “run-in” in their operating position, under loads which gradually increase to the maximum expected.

**THRUST LOADS**

As a result of the design of the Helical Gear tooth, an axial or thrust load is developed. Bearings must be adequate to absorb this load. The thrust load direction is indicated below. The magnitude of the thrust load is based on calculated Horsepower.

\[
\text{Axial Thrust Load} = \frac{126,050 \times \text{HP}}{\text{RPM} \times \text{Pitch Diameter}}
\]

Boston Helicals are all 45° Helix Angle, producing a tangential force equal in magnitude to the axial thrust load. A separating force is also imposed on the gear set based on calculated Horsepower.

\[
\text{Separating Load} = \text{Axial Thrust Load} \times 0.386
\]

Above formulae based on Boston 45° Helix Angle and 14-1/2° Normal Pressure Angle.

See page 114 for hardened and ground Thrust Washers.